

Characterisation Of Tiltrotor Blade Aerodynamic and Aeroelastic Behaviour In Propeller Mode At Stalled Conditions

Alexander D. Croke, Daniele Zagaglia, Richard B. Green George N. Barakos

James Watt School of Engineering, University of Glasgow

Glasgow, Scotland, UK

Outline of Presentation

- ♦ **Introduction**

- ♦ Research Motivation

- ♦ **Experimental Setup**

- ♦ United Kingdom National Rotor Rig (University of Glasgow)
- ♦ Test Matrix

- ♦ **Results**

- ♦ Blade Strain
- ♦ Performance Measurements
- ♦ Blade Tip Deflections

- ♦ **Conclusions**

Research Motivation



eVTOL

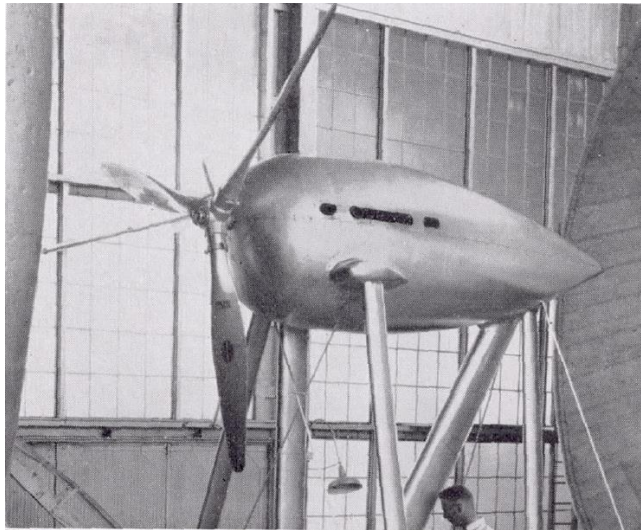


Tiltrotors

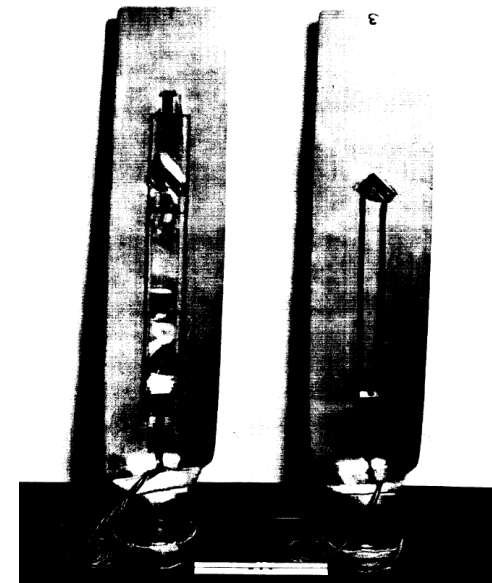
- ♦ Development of unconventional rotorcraft configurations
 - ♦ Bell V-280 tiltrotor to replace UH-60 for US Army
 - ♦ Rapid growth of civilian eVTOL market
- ♦ Tiltrotor blades are required to operate as propellers and rotors
 - ♦ Underexplored regions of the flight envelope
 - ♦ Newly developed thin and twisted shapes
- ♦ Further understand expected blade performance:
 - ♦ Aerodynamic boundaries (e.g blade stall)
 - ♦ Aeroelastic boundaries (e.g stall flutter)

Research Motivation

- ♦ Formation of modern aerodynamic and aeroelastic databases
 - ♦ Lack of experimental data for modern CFD validation
 - ♦ Clearly defined blade geometries and blade structural properties
 - ♦ Aid certification of new aircraft concepts
 - ♦ Utilise state-of-the-art experimental methods



Wooden blades with undefined structural properties (Hartman et al.)



Unoptimised blades instrumented with externally mounted strain gauges (Rogallo et al.)

Scope of Investigation

Undertake a multi-measurement experimental approach to characterise the aerodynamic performance and blade structural response of the MENtOR tiltrotor blades in propeller mode

- Carried out at University of Glasgow 9ft x 7ft DeHavilland wind tunnel
- Variation of rotational frequency (Ω), Advance ratio (J) and blade pitch at $0.75R$ ($\beta_{0.75R}$)
- Unsteady blade strain measurements obtained from instrumented blade set
- Thrust and torque data logged via Rotating Shaft Balance (RSB)
- Blade tip deflections measurements obtained via stereoscopic Digital Image Correlation (DIC)

UK National Rotor Rig

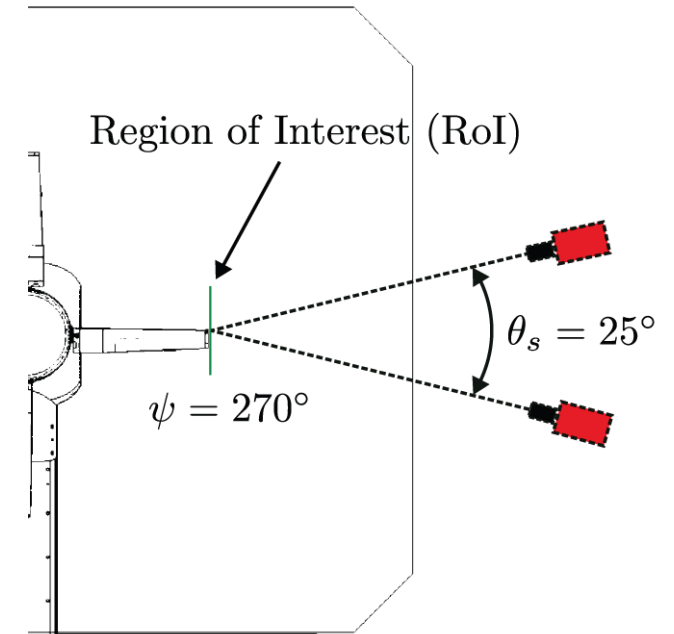
- Recently modified to fixed pitch propeller rig
- 1.25m diameter up to tip Mach-scaled conditions ($M_{tip} \approx 0.6$ at 3000 RPM)
- Large available power excess to test in deeply stalled conditions
- Telemetry system to transmit blade strain gauge and RSB data from rotating to stationary frame



Maximum Operational Limit	Quantity	Units
Thrust	3400	N
In-Plane Forces	550	N
Torque	350	Nm
In-Plane Moments	250	Nm
Available Power	125	kW
Rotational Frequency	3000	RPM
Pitch Angle Range	[-5,40]	°

Digital Image Correlation

- ♦ Stereoscopic imaging to assess blade tip deflections
 - ♦ Stereo angle of $\theta_s = 25^\circ$
 - ♦ Phase locked at $\varphi = 270^\circ$ using Hall-effect sensor
- ♦ Randomised speckle dot pattern applied to blade tip
 - ♦ Bending displacement and torsional twist obtained via image correlation
- ♦ Optical parameters
 - ♦ 5 Mpixel GigE-Prosilica CMOS cameras
 - ♦ 200 images per test condition
 - ♦ Magnification factor = 7.35 px/mm



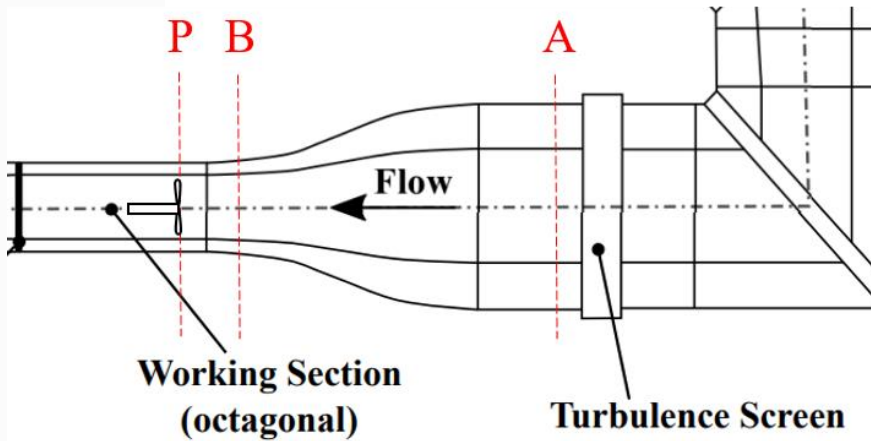
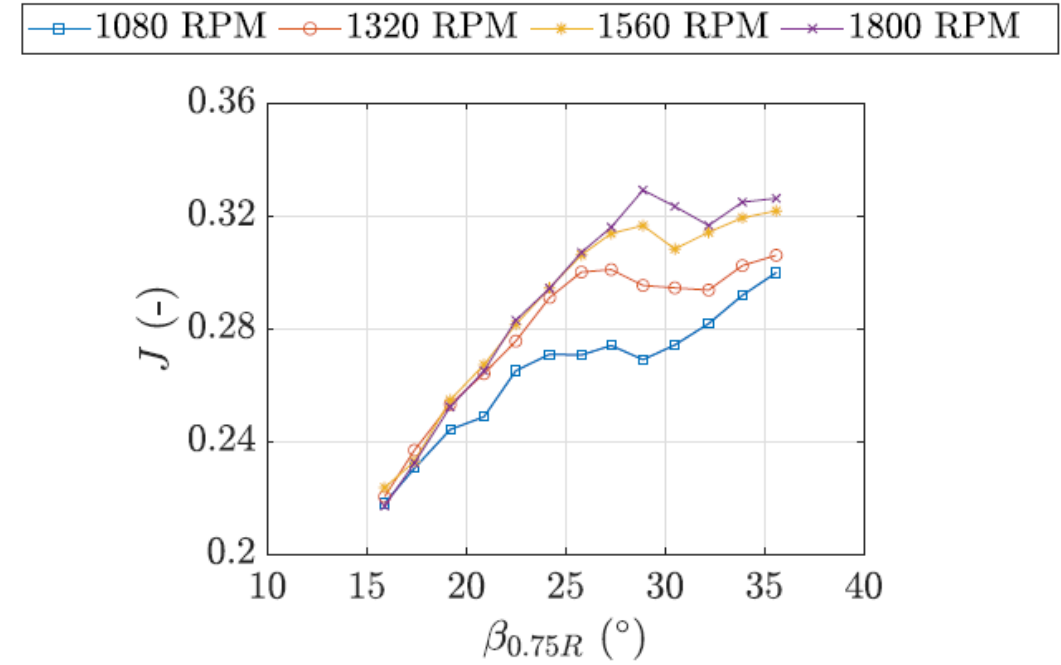
Test Matrix

- Three parameters:
 - Rotational frequency, $\Omega = 1080$ to 1800 RPM, $\Delta\Omega = 120$ RPM
 - Advance ratio, $J = \frac{U_\infty}{nD}$
 - Blade pitch, $\beta_{0.75} = 16^\circ$ to 35.7°

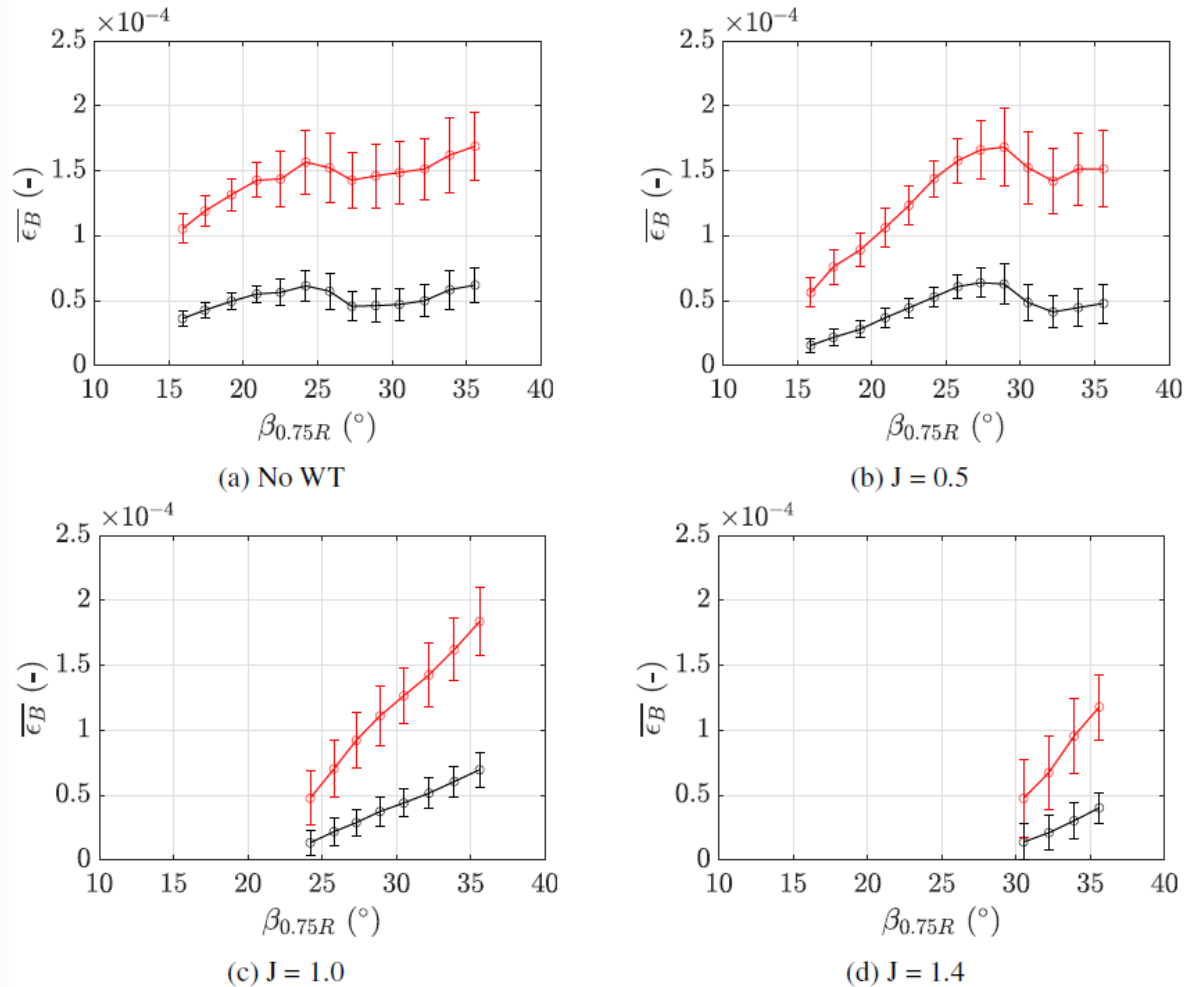
$\beta_{0.75R}$	No WT	J=0.5	J=0.7	J=0.9	J=1.1	J=1.3
16°	X	X				
19.1°	X	X	X			
22.3°	X	X	X	X		
25.5°	X	X	X	X	X	
28.7°	X	X	X	X	X	X
32°	X	X	X	X	X	X
35.7°	X	X	X	X	X	X

Results - Wind Tunnel Velocity Reference

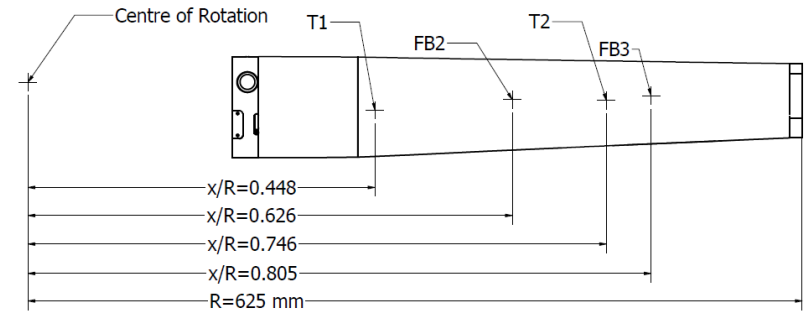
- Closed return wind tunnel
- Max $U_{\infty} = 70$ m/s, empty test section, $J = \frac{U_{\infty}}{nD}$
- Propeller induces velocity around the loop
- Calibration of contraction ring to set wind speed
- UKNRR rig acts as secondary fan



Results – Strain Gauge

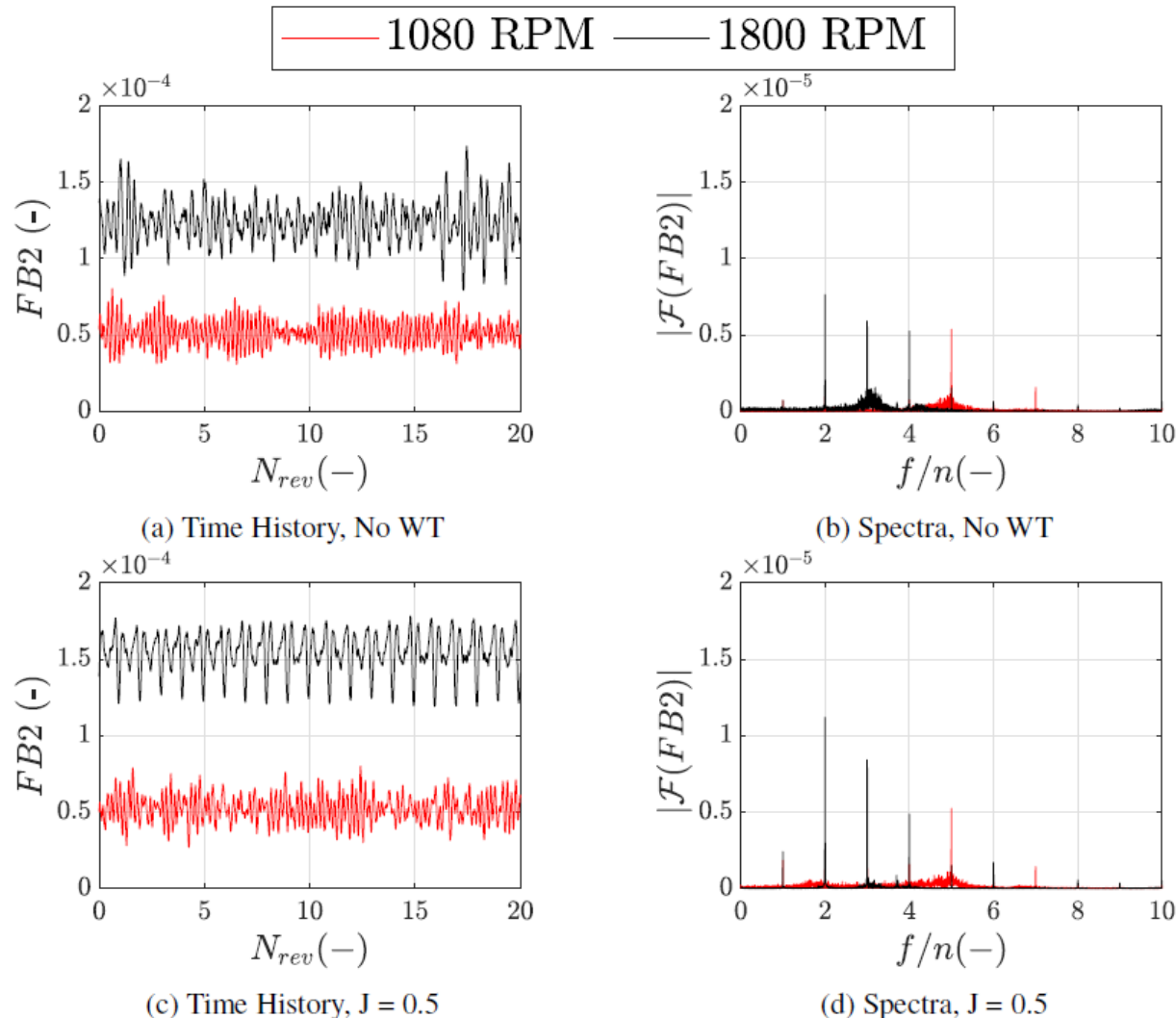
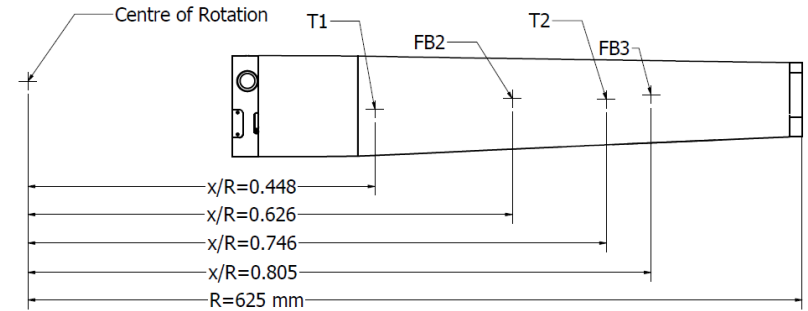


—○— FB2 —○— FB3



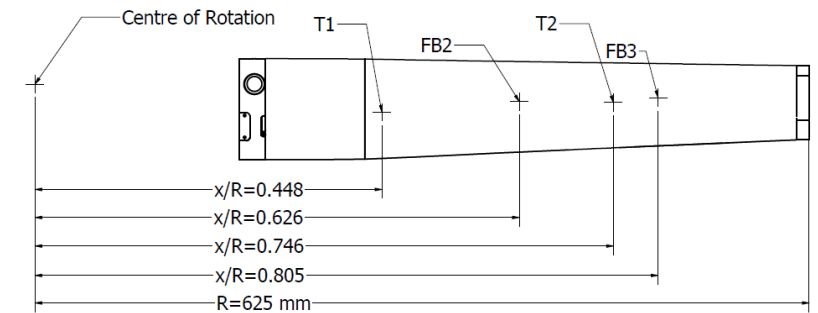
- $\Omega = 1080 \text{ RPM}$
- **Compensated strain** $\overline{\epsilon}_B = \epsilon_B \frac{\bar{\rho}}{\rho} \left(\frac{\bar{\Omega}}{\Omega} \right)^2$
 - $\bar{\rho} = 1.225 \frac{\text{kg}}{\text{m}^3}$
 - $\bar{\Omega} = 1800 \text{ RPM}$
 - Equivalent dynamic pressure
- Stall indicators
 - Deviation from linear behaviour
 - Growth in standard deviation post stall

Results – Strain Gauge

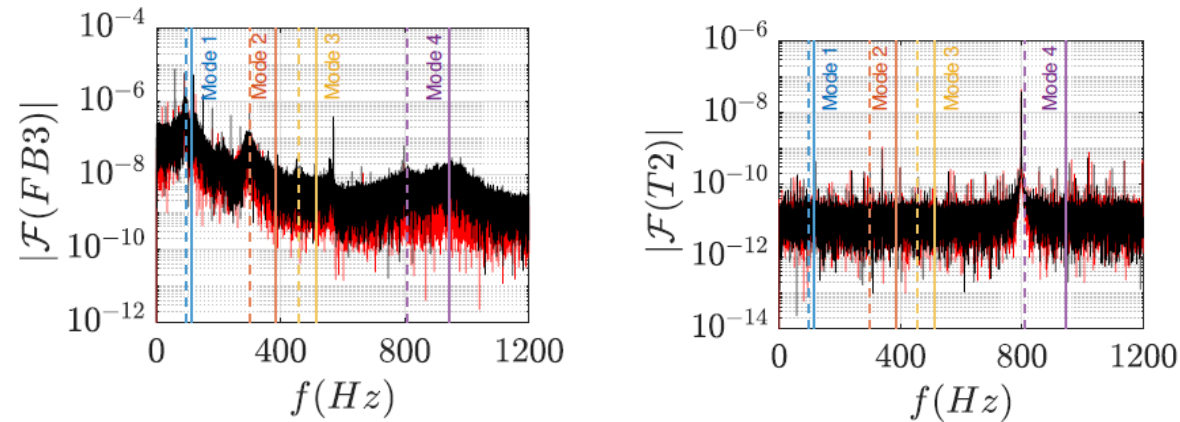


- $\beta_{0.75} = 30.5^\circ$
- No WT case shows presence of stall
 - Irregular pattern and amplitude
 - Distribution of signal energy across broadband of frequencies
- J = 0.5 shows stall at 1080 RPM
 - 1800RPM has repeatable pattern of constant amplitude with dominant 1/rev spike
 - FFT of 1800 RPM shows only harmonics

Results – Blade Modes



— 1080 RPM — 1800 RPM

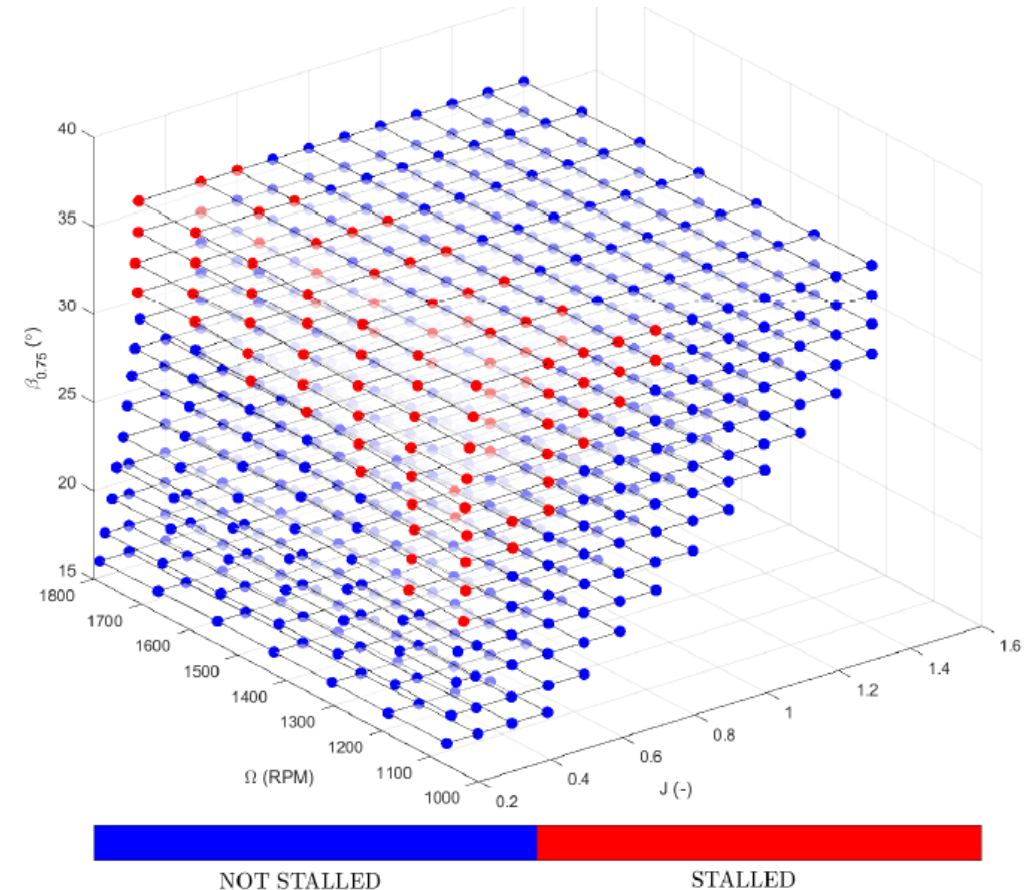


Mode Shape	Mode Number	Range of f_m (Hz)
1st Flap Bending	1	90-100
2nd Flap Bending	2	295-305
3rd Flap Bending	3	452-462
1st Torsion - Flap Bending	4	804-814

- SG data plotted using logarithmic scale
- Solid lines (Numerical) and dashed (Experimental)
- Blade modes identified
 - 1st peak at 90-100 Hz which translates to non-dimensional frequency of 3/rev @ 1800 RPM (30Hz) and 5/rev @ 1080 RPM (18Hz)
 - Stall cell shedding manifest as excitation of 1st flap bending mode
 - Blade torsional excitation needed to a excite a modal response such as stall flutter is large for rigid blades

Results - Stall Boundary Criteria

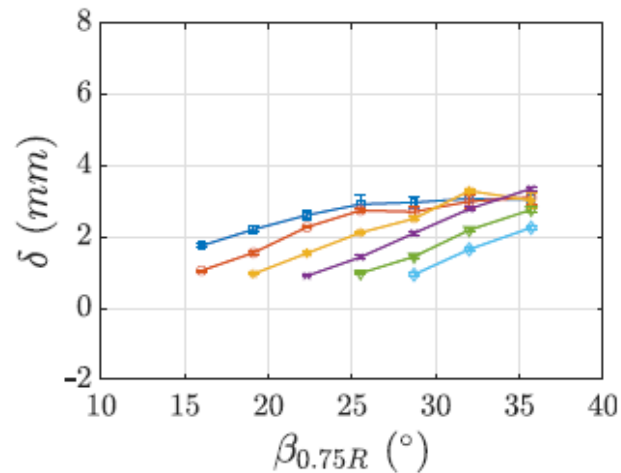
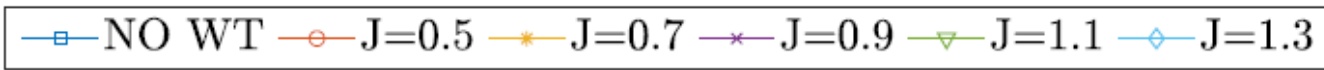
- ♦ **Collapse of induced velocity** around WT loop
- ♦ **Departure from linear behaviour** of the flap bending strain vs blade pitch curves
- ♦ Marked **increase of the standard deviation** of the flap bending strain, up to twice the pre-stalled conditions
- ♦ Presence of **non-harmonic content in the strain spectra**, up to 20 % in amplitude of the corresponding harmonic content
- ♦ **Non consistent oscillation amplitude** in the strain time history



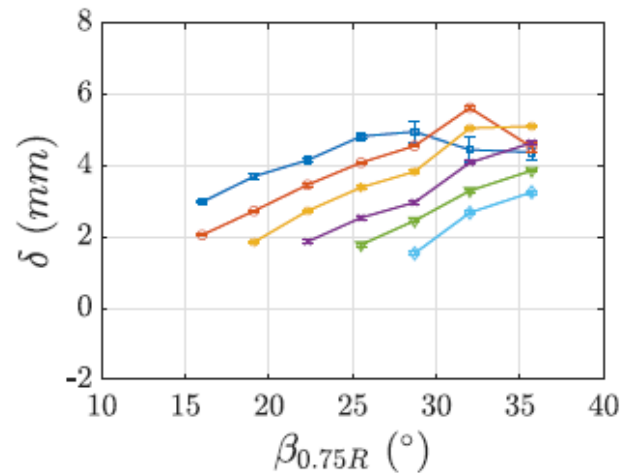
Slide Redacted

Results – Blade Tip Deflections

- Blade tip bending increased for larger RPM values due to higher loading.



$\Omega = 1080$ RPM



$\Omega = 1320$ RPM

- Flap bending deflections can be used to identify presence of stall
 - Reduction in deflection magnitude
 - Increased measurement unsteadiness represented via error bars.

Conclusions

- ♦ Stall onset was identified using a criteria established based upon the behaviour of the following parameters:
 - ♦ Induced velocity
 - ♦ Bending strain vs blade pitch curves
 - ♦ Standard deviation of strain gauge measurements
 - ♦ Strain spectra and time histories
- ♦ Performance measurements indicated the presence of stall at conditions identified using strain gauge identification criteria
- ♦ Flap bending tip deflections can also be utilised to identify stall onset
- ♦ Blade eigenmode frequencies can be identified from strain measurements

Thank you for your attention

Any Questions?

Alexander D. Croke

Research Assistant – University of Glasgow

Alexander.Croke@Glasgow.ac.uk